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P7030-1Attorney Docket No. 6340-000076/NP

A WHEEL BEARING AND A BEARING APPARATUS FOR A WHEEL OF VEHICLE OF <u>THE</u> SEMI-FLOATING TYPE HAVING THE WHEEL BEARING

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This application is a National Stage of International Application No. PCT/JP2004/017917, filed December 2, 2004, which claims priority to Japanese Patent Application No. 2003-411307, filed December 10, 2003 and Japanese Patent Application No. 2004-273178, filed September 21, 2004. The disclosures of the above applications are incorporated herein by reference

FIELD

[0001]

[0002] The present invention disclosure relates to a vehicle wheel bearing apparatus for a wheel of vehicle forto rotatably supporting support the wheel relative to a suspension apparatus of the vehicle, and, more particularly, to a wheel bearing wherein which a driving wheel is supported by a double row rolling bearing and a vehicle bearing apparatus for a wheel of vehicle of of a semi-floating type having the wheel bearing.

BACKGROUND

[-0002]

[0003] In a vehicle such as a truck, which has having a body of a frame structure construction, an axle structure of a driving wheel of a full-floating type has been widely adopted. In a recent driving wheel supporting structure, there has been widely adopted a unit structure of a double row rolling bearing has been widely adopted so as to improve the readiness of assembly, reduction of weight and size.

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One example of such a <u>prior art vehicle wheel</u> bearing apparatus for a wheel of vehicle of the prior art is shown in Fig. 4.

[0003]

[0004] In this vehicle wheel bearing apparatus for a wheel of vehicle, a drivingdrive shaft 52, connected to a differential apparatus (not shown), is inserted into an axle housing 51. A and a double row conical roller bearing 53 is mounted on the axle housing 51. A hub wheelwheel hub 54, rotatably supported by the double row conical roller bearing 53, is connected to a flange 56, via hub bolts 55. A pair of inner ringrings 57 are connected to each other by a connecting ring 58. The rings 57 are and fitted fit onto the end of the axle housing 51 and then securely fastened by a fastening nut 59. On the other hand, an outer ring 60, of the double row conical roller bearing, is fitted fit into the hub wheelwheel hub 54. The outer ring 60 is and axially secured with its both its ends being sandwiched by the flange 56 of the driving drive shaft 52 and a brake rotor 61. A double Double row conical rollers 62 are rollably contained by cages 63 between the annular space between the inner and outer rings 57 and 60, and soals Seals 64 are arranged at both ends of the annular space to seal off the inside of the wheel bearing eff from the outside.

[0004]

[0005] The inboard side end of the inner ring 57 is formed with an annular stepped portion 65. A on which a seal ring 66 is mounted on the stepped portion 65. An annular recess 67 is formed on the outer circumferential surfaces at mutually abutted abutting portions of the pair of inner rings 57. A and a seal ring 68, of an elastic material member, is fitted therein into the recess 67. These The seal rings 66

and 68 prevent penetration or ingress of rain water or dusts into the axle housing 51, leakage of differential gear oil to outside and ingress of the differential gear oil into the inside of the bearing (see Japanese Laid-open Patent publication No. 99172/2001).

Disclosure of the Invention
Problems to be solved by the Invention
[0005]

<u>[0006]</u> However, since the <u>prior art vehicle wheel</u> bearing apparatus for a wheel of vehicle of the prior art has a structure <u>sosuch</u> that the double row of conical roller bearing 53 is arranged between the <u>hub wheelwheel hub</u> 54 and the axle housing 51, and that the <u>drivingdrive</u> shaft 52 is inserted into the axle housing 51, and then the flange 56 of the <u>is drivingdrive</u> shaft 52 is connected to the <u>hub wheelwheel hub</u> 54 by the hub bolts 55, <u>a</u> reduction of the weight and size of the bearing apparatus is limited. <u>Also, as well as assembly</u> of the bearing apparatus is complicated <u>due to by requirement of a large number of structural parts.</u>

SUMMARY OF THE INVENTION

_[0006]

[0007] It is therefore an object of the present invention disclosure to provide a vehicle wheel bearing apparatus for a wheel of vehicle which can reduce the weight, size and a number of parts. Also, the bearing apparatus and also can prevent prevents ingress of rain water or dusts and leakage of differential gear oil.

<u>[0008]</u> For achieving the object, there is provided, according to the <u>The</u> present invention <u>disclosure provides a vehicle wheel</u> of claim 1, a bearing apparatus for a wheel of vehicle structured as a unit of a <u>hub wheel wheel</u> and a double row

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rolling bearing which comprisescomprising an inner member including with a hub wheelwheel hub integrally formed with a wheel mounting flange on its one end. with a wheel mounting flange and A serration is formed on its inner circumferential surface of the wheel hub. _with-a-serration-and-having-anAn axially extending cylindrical portion is formed with the flange. One or more and innerinner rings are pressfitted fit onto the cylindrical portion of the hub wheel wheel hub. At least one of inner raceway surfaces is and formed on whichan outer circumferential surface of the inner rings. -with at least one of inner raceway surfaces; anAn outer member is arranged around the inner member. The outer member includes and formed with double row outer raceway surfaces on its inner circumferential surface eppesitelyopposite to the inner raceway surfaces;. Double double row rolling elements are arranged between the inner and outer raceway surfaces of the inner member and the outer member. A-a cage for freely rollably holdingholds the rolling elements;. Seals and seals for sealingseal an annular space between the inner member and the outer member. A characterized in that a partition wall is integrally formed on the hub wheelwheel hub at its outboard side tofor closing close a central bore of the hub wheelwheel hub.

-[8000]

<u>[0009]</u> According to the present invention of claim 1, since Since the partition wall is integrally formed on the <u>hub wheelwheel hub</u> at its outboard side for closing to <u>close</u> a central bore of the <u>hub wheelwheel hub</u>, the rigidity of the <u>hub wheelwheel</u> <u>hub</u> is increased. Thus, and thus it is possible to suppress the elastic deformation of the <u>hub wheelwheel hub</u> and to improve the durability of the bearing apparatus.

[0009]

[0010] According to the present invention of claim 2, since Since said at least one of the inner raceway surfaces is formed directly on the outer circumferential surface of the hub wheel wheel hub, it is possible to further reduce the weight and size and weight and increase the rigidity of the bearing.

[0010]

[0011] According to the present invention of claim 3, since Since the end of saidthe cylindrical portion is plastically deformed radially outward to form a caulked portion forto preventing prevent the inner ring from being slippedslipping off offrom the cylindrical portion of the hub wheel wheel hub, it is unnecessary to control the amount of preload of the bearing as in the prior art by tightly fastening the inner ring using a nut. Thus, ease easiness of assembly of the bearing apparatus to a vehicle can be improved. Also, and the predetermined amount of preload can be kept for a long term. In addition, it is possible to substantially reduce the number of parts and to reduce the manufacturing cost, and the weight and size of the bearing due to the improvement inof the ease easiness of assemble assembly.

[0011]

[0012] Preferably according to the present invention of claim 4, since Since the outer circumferential region of the wheel mounting flange, from its inboard side base of inboard side to the cylindrical portion, is hardened by high frequency induction hardening as having the to have a surface hardness of 58~_64 HRC, and since the caulked portion is remained remains unhardened as having to have a the surface hardness of 25 HRC or less after forging, it is possible to improve the

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durability of the hub wheelwheel hub and the workability of the caulked portion during its plastic deformation. Thus, this improves and thus the reliability of and quality of the bearing.

[0012]

<u>I00131</u> According to the present invention of claim 5, since Since athe vehicle wheel bearing apparatus for a wheel of vehicle of thea semi-floating type emprising apparatus for a wheel of vehicle under a body of a vehicle; a hollow driving drive shaft inserted into the axle housing; and the vehicle wheel bearing apparatus for a wheel of vehicle of any one of claims 1~4 arranged between the driving drive shaft and an opening of the axle housing; and the drivedriving shaft connected to thesaid inner member so that a torque is transmittable therebetween, between the two, it is possible to provide a vehicle wheel bearing apparatus for a wheel of vehicle of the semi-floating type which has a high rigidity and can reduce the weight and size and also of the gearing apparatus. Also, this can prevent the leakage of differential gear oil to the outside as well as the ingress of rain water or dusts from the outside into the differential gear oil through the driving shaft.

[0014] — According to the present invention of claim 6, since Since the driving drive shaft is separably connected to the inner member via the serration, the workability of assembly of the bearing apparatus can be remarkably improved.

Effect of the Invention

-[0014]-

[0015] According to the <u>present vehicle wheel</u> bearing apparatus for a wheel of vehicle of the present invention, since the <u>vehicle wheel</u> bearing apparatus for a

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wheel-mounting-flange-and enwith its inner circumferential surface_including-with a serration and having an axially extending cylindrical portion withand one or more inner rings press-fittedfit onto the cylindrical portion of the hub-wheelwheel hub and with the inner rings formed with at least one of inner raceway surfaces on whichits outer circumferential surface; with at least one of inner raceway surfaces; an outer member arranged around the inner member and formed with double row outer raceway surfaces on its inner circumferential surface eppositelyopposite to the inner raceway surfaces; double row rolling elements arranged between the inner and outer raceway surfaces of the inner member and the outer member; a cage for-freely rollably holding the rolling elements—(5); and—seals for sealing an annular space between the inner member and the outer member; and is characterized in that a partition wall is-integrally formed on the hub-wheelwheel hub at its outboard side forto close closing off a central bore of the hub-wheelwheel hub, the rigidity of the hub-wheelwheel hub is increased and thus it is possible to suppress the elastic

wheel of vehicle-is structured as a unit of a hub wheel wheel hub and a double row

rolling bearing comprises comprising an inner member withincluding a hub

wheelwheel hub integrally formed with a wheel mounting flange on its-one end with a

[0015]

apparatus.

[0016] In addition, according to the <u>present vehicle wheel</u> bearing apparatus for a wheel of vehicle of the present invention, since a <u>vehicle wheel</u> bearing apparatus for a wheel of vehicle of a semi-floating type comprising an axle housing

deformation of the hub wheelwheel hub and to improve the durability of the bearing

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supported under a body of vehicle; a hollow driving shaft inserted into the axle housing; and the vehicle wheel bearing apparatus for a wheel of vehicle of any one of claims 1~4is arranged between the drivingdrive shaft and an opening of the axle housing; and the drivingdrive shaft is connected to thesaid inner member so that a torque is transmittable therebetween between the two, it is possible to provide a vehicle wheel bearing apparatus for a wheel of vehicle of the semi-floating type which has a high rigidity and can reduce the weight and size. Also, it prevents and also can prevent the leakage of differential gear oil to the outside as well as the ingress of rain water or dusts from the outside into the differential gear oil through the drivingdrive shaft.

Best mode for carrying out the Invention - [0016]

[0017] According to the present invention, there is provided a A vehicle wheel bearing apparatus is for a wheel of vehicle-structured as a unit of a hub-wheelwheel hub and a double row rolling bearing comprising which comprises an inner member including with a hub-wheelwheel hub integrally formed with a wheel mounting flange on its one end. with a wheel mounting flange A serration is formed and on its inner circumferential surface of the wheel hub. Also, with a serration and having an axially extending cylindrical portion is formed with the flange. One or more i-and inner-nner rings are press-fitted fit onto the cylindrical portion of the hub-wheelwheel hub. At least one inner raceway surface is—and formed on anwhich outer circumferential surface of the inner rings. An-with at least one of inner raceway surfaces; an outer member is arranged around the inner member and formed with double row outer raceway surfaces on its inner circumferential surface oppositely opposite to the inner

raceway surfaces; <u>Double double</u> row rolling elements <u>are</u> arranged between the inner and outer raceway surfaces of the inner member and the outer member. A; a cage for freely rollably <u>holdingholds</u> the rolling elements. <u>Seals seal</u>; and seals for sealing an annular space between the inner member and the outer member. A; characterized in that a partition wall is integrally formed on the <u>hub wheelwheel hub</u> at its outboard side for closingto close a central bore of the <u>hub wheelwheel hub</u>.

BRIEF DESCRIPTION OF THE DRAWINGS

[0018] Additional advantages and features of the present invention disclosure will become apparent from the subsequent description and the appended claims, taken in conjunction with the accompanying drawings, wherein:

[0019] Fig. 1 is a longitudinal-section view of a first embodiment of a <u>vehicle</u> wheel bearing apparatus for a wheel of vehicle of the present invention;

[0020] Fig. 2 is a partially enlarged longitudinal-section view of Fig. 1 showing a wheel bearing;

[0021] Fig. 3 is a longitudinal-section view of a second embodiment of the vehicle wheela bearing apparatus for a wheel of vehicle of the present invention; and

[0022] Fig. 4 is a longitudinal-section view of a <u>prior art vehicle wheel</u> bearing apparatus for a wheel of vehicle of the prior art.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

[0023] Preferred embodiments of the present invention disclosure will be described with reference to accompanied drawings.

First embodiment

[0017]

[0024] Fig. 1 is a longitudinal-section view of a first embodiment of a bearing apparatus for a wheel of the present invention disclosure., and Fig. 2 is a partially enlarged longitudinal-section view of Fig. 1. In the description of the present invention disclosure, a side of a bearing positioned outward a of the vehicle, when it is mounted on thea vehicle, is referred to as the "outboard" side (the left side in a drawing). A side, and a side inward of thea vehicle is referred to as the "inboard" side (the right side in a drawing).

[0018]

[0025] In a vehicle wheel bearing apparatus for a wheel of vehicle of the present invention disclosure, a hub wheel wheel hub 1 and a double row rolling bearing 2 are formed as a unit and connected to a driving drive shaft "D/S". The double row rolling bearing 2 includes comprises an inner member 3, an outer member 4, and a double row rolling elements (tapered rollers) 5 freely rollably contained between the inner and outer members 3 and 4. The inner member 3 includes the hub wheel wheel hub 1 and a pair of inner rings 10 press-fitted fit onto the hub wheel wheel hub 1. The hub wheel wheel hub 1 is integrally formed, at its outboard side, with a wheel mounting flange 6. A on which, a wheel "W" and a brake rotor "B" are mounted onto the wheel mounting flange 6. and from which an An axially extending cylindrical portion 7 extends from the wheel mounting flange 6. An inner circumferential surface (bore) of the hub wheel wheel hub 1 is integrally formed with a serration (or spline) 8. into which and serrated portion of the driving drive shaft "D/S" is inserted into the bore so that a torque can be transmitted

therebetween between the two. A and with a partition wall 9, on the outboard side of the hub wheelwheel hub 1, closes for closing a central bore (inner circumferential surface) of the hub wheelwheel hub 1.

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[0019]

[0026] As shown in Fig. 2, the double row rolling bearing 2 comprises includes an outer member 4 formed with double row outer raceway surfaces 4a on its inner circumferential surface. A and with a body mounting flange 4b, formed on an outer circumferential surface of the outer member, is to be secured on an axle housing "H". on its outer circumferential surface, a pair of inner rings 10A pair of inner rings 10 is inserted in the outer member 4. The inner rings 10 are and formed with double row tapered inner raceway surfaces 10a on itstheir outer circumferential surface oppositelyopposite to the outer raceway surfaces 4a,. Double double row rolling elements 5 are arranged between the inner and outer raceway surfaces 10a, 4a. , and aA cage 11 for freely rollably holdingholds the rolling elements 5. Each of the inner rings 10 is formed with, at its larger diameter end, a large flange 10b for guidingto guide the rolling elements 5. The pair of inner rings 10 are is arranged so that their inner ends are abuttedabut each other and thus form form a so-called a back-abutted type double row tapered roller bearing. Seals 12are arranged at either ends of the outer member 4 to seal an annular space between the outer member 4 and the inner rings 10. These The seals 12 prevent both penetration of rain water or dusts from the external circumstances and leakage of lubricating grease sealed within the bearing. The inboard side seal 12 further prevents penetration or ingress of differential gear oil into the inside of the bearing passing through the serration 8 of the hub wheelwheel hub 1.

-[0020]-

[0027] The pair of inner rings 10 areis press-fittfited onto the cylindrical portion 7 of the hub wheelwheel hub 1. The inner rings 10 and are prevented from being-axially slippedslipping off from the cylindrical portion 7 by a caulked portion 13. The caulked portion 13 is formed by plastically deforming the end of the cylindrical portion 7 radially outward. Since this embodiment adopts athe self-retaining structure of the second generation, it is not required to control an amount of preload as in a conventional manner by tightly fastening a nut against the inner ring. Accordingly, it is possible to substantially reduce the number of parts and thus to improve the readiness of assembly as well as to reduce its manufacturing cost, sizeweight and weightsize.

[0021]

[0028] The hub-wheelwheel hub 1 is made of medium carbon steel such as S53C which includes including carbon of about 0.40~_0.80% by weight. The wheel hub 1 is and hardened by high frequency induction quenching so that the inboard base side of the wheel mounting flange 6 at its inboard side and the cylindrical portion 7 of the hub wheelwheel hub 1 have athe surface hardness of 58~_64 HRC (the hardened portion is shown in the drawings by cross-hatched lines). The caulked portion 13 is remained remains as an unhardened portion with having its surface hardness of 25 HRC or less. This improves the durability and workability of the caulked portion 13 and also prevents the generation of cracks-therein.

-[0022]

[0029] The outer member 4 is also made of medium carbon steel such as S53C which including includes carbon of about 0.40~_0.80% by weight. The and the double row outer raceway surfaces 4a and the inner circumferential surface of the outer member 4 whereon which the seal 12 is mounted are hardened by high frequency induction quenching so that their surface hardness is within 58~_64 HRC. On the other hand, the inner rings 10 isare made of high carbon chrome bearing steel such as SUJ2. _-and-The inner rings 10 are hardened to itstheir core by dippingdip quenching to have thea surface hardness of HRC 58~64. Although, _-it the bearing is herein-illustrated as a double row tapered roller bearing using tapered roller as the rolling elements 5, athe double row angular ball bearing, using balls, may be also used.

[0023]

[0030] In this embodiment, since the partition wall 9 is integrally formed at the outboard side on the hub wheelwheel hub 1 at its outboard side for closingto close the central bore (inner circumferential surface) of the hub wheelwheel hub 1, high rigidity of the hub wheelwheel hub 1 can be maintained high even though although it is used in the bearing apparatus of the semi-floating type. Thus, and thus it is possible to suppress an elastic deformation of the hub wheelwheel hub 1 althougheven though awhen the moment load is applied to the hub wheelwheel hub 1 during running of thea vehicle. It and also to surely prevent prevents ingress of rain water or dusts from the ambient circumstances into the drivingdrive shaft "D/S" and thus into the differential gear oil.

Second embodiment

-[0024]

[0031] Fig. 3 is a longitudinal-section view of a second embodiment of athe vehicle wheel bearing apparatus for a wheel of vehicle of the present invention disclosure. Since difference of this embodiment from the first embodiment only resides in the structure of the hub wheel wheel hub, the same numerals are used as those used in the first embodiment for designating to designate the same structural elements.

[0025]

structured as a unit of athe hub wheelwheel hub 14 and a double row rolling bearing 15. The double row rolling bearing 15 emprises includes an inner member 16, an outer member 4, and a double row rolling elements 5 and 5 freely rollably contained between the inner and outer members 16 and 4. The inner member 16 includes the hub wheelwheel hub 14. An and the inner wheelring 10 is press-fitted fit onto the hub wheelwheel hub 14. The hub wheelwheel hub 14 is integrally formed, at its outboard side, with a wheel mounting flange 6. On which, a wheel (not shown in Fig. 3) is mounted on the flange 6. An and with an inner raceway surface 14a is formed on the wheel hub 14 at of the outboard side of the bearing 15. and has the The cylindrical portion 7 axially extending extends from the inner raceway surface 14a. The hub wheelwheel hub 14 is formed with a serration (or spline) 8 on its inner circumferential surface (bore). with a serration (or spline) 8 into which a serrated portion of the driving drive shaft (not shown in Fig. 3) is inserted into the bore to transmit so that a torque can be transmitted therebetween between the two. and with

a_A partition wall 9, on the outboard side of the hub_wheelwheel hub 14, for closingcloses a central bore (inner circumferential surface) of the hub_wheelwheel hub 14.

[0026]

[0033] The outer circumferential surface of the hub—wheelwheel hub—14 is formed with a flange portion 14b corresponding to the large flange 10b of the inner ring 10. A-and-a stepped portion 14c is in the outer circumferential face to abut which-an inner end face (smaller end face) of the inner ring 10 abute. Thus, this provides athe so-called back-abutted type double row tapered roller bearing is structured structure. In addition, the inner ring 10 is press-fitted fit onto the cylindrical portion 7 of the hub—wheelwheel hub—14. The inner ring 10 and is prevented from being-axially slipped slipping off from the cylindrical portion 7 by a caulked portion 13. The caulked portion 13 is formed by plastically deforming the end of the cylindrical portion 7 radially outward. Since this embodiment adopts athe self-retaining structure of such-a third generation, it is not required to control an amount of preload as in a manner similar to the first embodiment by tightly fastening a nut against the inner ring. Accordingly, it is possible to improve the readiness of assembly as well as to maintain the amount of preload for a long term.

-[0027]-

[0034] Since the inner raceway surface 14a is directly formed on the outer circumferential surface of the hub wheelwheel hub 14 and the partition wall 9 is also integrally formed on the hub wheelwheel hub 14 at its outboard side end, the rigidity of the hub wheelwheel hub 14 is increased. Accordingly, it is possible to suppress an

elastic deformation of the hub wheelwheel hub 14 althougheven though the moment load is applied to the hub wheelwheel hub 14 during running of thea vehicle. Also, it and also to surely prevent prevents ingress of rain water or dusts from the ambient circumstances into the drivingdrive shaft "D/S" and thus into the differential gear oil.

Applicability in industry

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[0035] The present vehicle wheel bearing apparatus for a wheel of vehicle of the present invention can be applied to a bearing apparatus for a wheel of <u>a</u> vehicle of the driving wheel side of the semi-floating type in which where a wheel bearing is arranged in opened portions between a driving drive shaft and aan axle housing.

[0036] [0029] The present invention disclosure has been described with reference to the preferred embodiment. Obviously, modifications and alternations will occur to those of ordinary skill in the art upon reading and understanding the preceding detailed description. It is intended that the present invention disclosure be construed as including to include all such alternations and modifications insofar as they come within the scope of the appended claims or the their equivalents thereof.

WHAT IS CLAIMED IS:
1A <u>vehicle wheel</u> bearing apparatus for a wheel of vehicle structured as
a unit of a hub wheelwheel hub (1, 14) and a double row rolling bearing (2, 15)
comprising:
an inner member (3, 16) including a hub wheelwheel hub (1, 14)
integrally formed with a wheel mounting flange on its one end, with a wheel
mounting flange (6) and on itsan inner circumferential surface of the wheel hub is
formed with a serration, (8) and having an axially extending cylindrical portion with a
pair of inner raceway surfaces extend from the flange (7);
and_one or more inner rings (10) rings being press-fittedfit onto the
cylindrical portion (7) of the hub wheelwheel hub, the one or more inner rings are
formed with at least one of the inner raceway surfaces (1, 14) and formed on
itswhich outer circumferential surface with at least one of inner raceway surfaces
(10a) ;
an outer member (4)is arranged around the inner member, the outer
member is (3, 16) and formed with double row outer raceway surfaces (4a) on its
inner circumferential surface oppositely opposite to the inner raceway surfaces-(10a);
double row rolling elements (5) are arranged between the inner and
outer raceway surfaces (10a, 4a) of the inner member (3, 16) and the outer member
(4) ;
a cage (11) for freely rollably holdingholds the rolling elements (5) ;
and
seals (12) for sealingseal an annular space between the inner

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member (3, 16) and the outer member (4); and

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	racterized in that	-a partition wall (9)-is integrally formed on the
hub wheelwheel hub	(1, 14) at its out	board side for closingto close a central bore of
the hub wheel wheel	<u>hub (1, 14)</u> .	
2A <u>The</u> v	<u>rehicle wheel</u> bea	ring apparatus for a wheel of vehicle of claim 1
wherein said at lea	st one (14a) -of_	said inner raceway surfaces (10a)-is formed
directly on the outer	circumferential su	rface of the hub wheelwheel hub (1, 14).
3A <u>The</u> v	<u>rehicle wheel</u> bea	ring apparatus for a wheel of vehicle of claim 1
wherein the end of s	aid cylindrical por	tion (7)-is plastically deformed radially outward
to form a caulked	portion (13) for	preventing the inner ring (10)—from being
slippedslipping off o	from the cylindri	cal portion (7) of the hub wheelwheel hub (1,
14) .		
4A <u>The</u> v	<u>rehicle wheel</u> bea	ring apparatus for a wheel of vehicle of claim 3
wherein thean outer	circumferential re	egion of the wheel mounting flange (6)-from its
base of an inboard e	idebase side to	the axially extending cylindrical portion (7) is
hardened by high f	requency induction	on hardening <u>toas having have</u> thea surface
hardness of about 5	8 <u>~~</u> 64 HRC, and	d the caulked portion (13) is remained remains
unhardened as hav	ng-to have athe	surface hardness of 25 HRC or less after
forging.		·

ABSTRACT OF DISCLOSURE

An object of the present invention is to provide a vehicle wheel bearing apparatus for a wheel of vehicle-which can reducereduces the weight, size and a number of parts and also can prevent prevents ingress of rain water or dusts and leakage of differential gear oil. According to the present invention there is provided a bearing apparatus for a wheel of vehicle is structured as a unit withof a hub wheelwheel hub (1,14) and a double row rolling bearing (2, 15). -comprising anAn inner member (3, 16) includingincludes athe hub wheelwheel hub (1, 14) integrally formed with a wheel mounting flange (6) on its one end. with a wheel mounting flange (6) and on its The wheel hub -inner circumferential surface with includes a serration (8). An and having an axially extending cylindrical portion (7) extends from the flange (6).7 One or more and inner rings (10) are press-fitted fit onto the cylindrical portion (7) of the hub wheelwheel hub (1, 14). The inner rings are and formed on whichtheir outer circumferential surface with at least one of the inner raceway surfaces (10a);. -anAn outer member (4) is arranged around the inner member (3, 16). The outer member (4) is and formed with double row outer raceway surfaces (4a) on its inner circumferential surface oppositelyopposite to the inner raceway surfaces (10a). Double; double row rolling elements (5) are arranged between the inner and outer raceway surfaces (10a, 4a) of the inner member (3, 16) and the outer member (4);. A-a cage (11) for-freely rollably holdingholds the rolling elements (5). Seals; and seals (12) for sealingseal an annular space between the inner member (3, 16) and the outer member (4). A-characterized in that a partition wall (9) is integrally formed on the hub wheel wheel hub (1, 14), at its outboard side, forto close closing a central bore of the hub wheelwheel hub (1, 14).